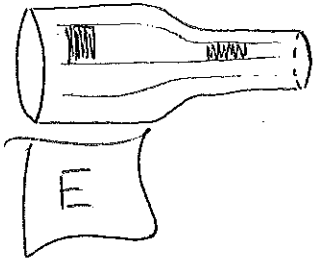


①



② The ship will sit at the point when the height when the volume of the ship that is underwater times the density equals the mass of the ship.

Since ship B is narrower at the bottom, ~~the more of it~~ a larger percent of it must sit <sup>below</sup> above water to achieve the same volume  
 $\Rightarrow$  B rides lower  $\iff$  A rides higher  $\Rightarrow$  A

③ The solutions to an equation of the form:

$$m \frac{d^2x}{dt^2} + b \frac{dx}{dt} + kx = 0$$

is given by:

$$\left( e^{-bt/2m} \right) e^{\pm \sqrt{b^2/4m - k/m}}$$

when the system is ~~over-damped~~ under-damped;

$$\frac{b^2}{4m} - k/m < 0$$

So all damping is determined by:

$$e^{-bt/2m} \Rightarrow \text{only } b, m \text{ affect damping time} \Rightarrow C$$

④ Fluids are not on test.  $\checkmark \checkmark \checkmark$

we know:  $A_1 v_1 = A_2 v_2$  (continuity)

$$\frac{1}{2} \rho v_1^2 + P_1 = \frac{1}{2} \rho v_2^2 + P_2$$

$$\Rightarrow \frac{1}{2} \rho v_1^2 + P_1 = \frac{1}{2} \rho \left(\frac{A_1}{A_2}\right)^2 v_1^2 + P_2$$

$$\Rightarrow \frac{1}{2} \rho \left(1 - \frac{r_1^4}{r_2^4}\right) v_1^2 + P_1 = P_2 = \frac{1}{2} \rho \left(\frac{r_2^4 - r_1^4}{r_2^4}\right) v_1^2$$

Thus, if  $r_1 > r_2$ ,  $P_1 > P_2 \Rightarrow A, \beta, \phi$

It is not linear with respect to  $r_2 \Rightarrow \nabla$

After this it's tricky:

Graph F, looks like the graph of  $1/x^n$ , but this is  $1 - 1/x^n$

$\Rightarrow D$

⑤ For a perfect black-body, the power radiated is given by

$$P = AT^4$$

$\hookrightarrow$  area of the sphere.

There steady state power will be the same in each case

$$\Rightarrow A_1 T_1^4 = A_2 T_2^4$$

$$= \pi r_1^2 T_1^4 = \pi r_2^2 T_2^4$$

$$\Rightarrow r_1^2 T_1^4 = (2r_1)^2 T_2^4$$

$$\Rightarrow \frac{1}{4} T_1^4 = T_2^4$$

$$\Rightarrow T_2 = \frac{T_1}{\sqrt{2}}$$

$\Rightarrow \boxed{B}$

⑥ The specific heat of the substance is the amount of heat needed to change a unit mass of the substance by a unit temp, i.e.

$$\left(\frac{dQ}{dT}\right)\left(\frac{1}{m}\right) = \underbrace{\left(\frac{dQ}{dT}\right)}_{\substack{\text{rate at which heat is added} \\ \text{slope of the graph}}}\left(\frac{dT}{dT}\right)\left(\frac{1}{m}\right) = \left(\frac{dQ}{dT}\right) / \underbrace{\left(\frac{dT}{dT}\right)}_m$$

Thus, we have something of the form of C, D, and E. Since we want the specific heat of the solid, which is section A, we pick C

⑦ When you read "after a very long time" you should think "in equilibrium"  $\iff$  everything is the same temp.  $\implies$  C

⑧ Since what causes the water to come out was the pressure of all the water above it  $\iff$  the weight of the water above it, and since things are "weightless" in a frame of reference in free-fall, there is no pressure.  $\implies$  No water is pushed out  $\iff$  C

⑨

SHO  $\implies$

$$x(t) = A \cos(\omega t + \delta)$$

$$v_{\text{max}}(t) = -A\omega \sin(\omega t + \delta)$$

$$a_{\text{max}}(t) = -A\omega^2 \cos(\omega t + \delta)$$

$$\implies KE = \frac{1}{2} A^2 \omega^2 \sin^2(\omega t + \delta)$$

$$PE = \frac{1}{2} k A^2 \cos^2(\omega t + \delta)$$

$$T = \frac{2\pi}{\omega} = 2\pi \sqrt{\frac{m}{k}}$$

9) For a simple harmonic pendulum:

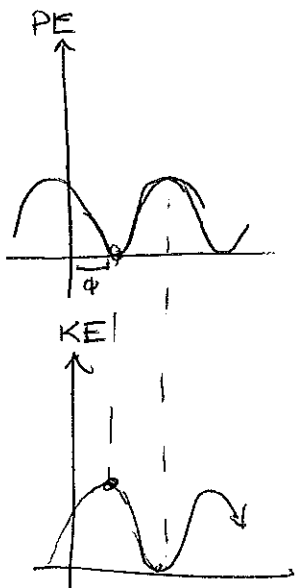
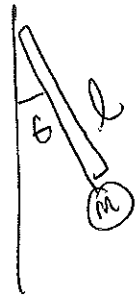
$$\theta(t) = \theta_{\max} \sin(\sqrt{g/L} t - \phi)$$

$$\frac{d\theta}{dt} = \omega(t) = \frac{v_{\tan}}{L} = \sqrt{g/L} \theta_{\max} \cos(\sqrt{g/L} t - \phi)$$

$$\frac{d^2\theta}{dt^2} = \alpha(t) = \frac{a_{\tan}}{L} = -\left(\frac{g}{L}\right) \theta_{\max} \sin(\sqrt{g/L} t - \phi)$$

$$\Rightarrow PE \propto \theta_{\max}^2 \sin^2(\sqrt{g/L} t - \phi)$$

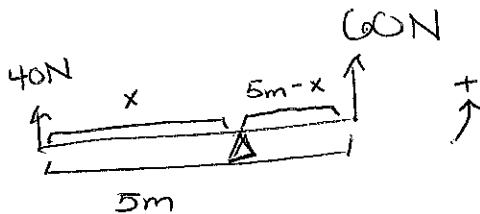
$$KE \propto \theta_{\max}^2 (g/L) \cos^2(\sqrt{g/L} t - \phi)$$



$$T = 2\pi \sqrt{L/g} \Rightarrow \text{D is wrong}$$



10) a)



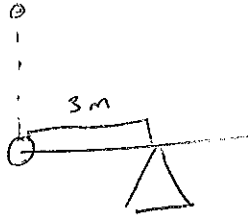
$$\sum \tau = (-40\text{N})(x) + (60\text{N})(5\text{m}-x) = 0$$

$$\Rightarrow -100\text{N}x + 300\text{Nm} = 0$$

$$\Rightarrow 300\text{Nm} + 100\text{N}(x) = 300\text{Nm}$$

$$\Rightarrow \boxed{x = 3\text{m}}$$

10) (cont.) b) ANGULAR MOMENTUM WILL NOT BE ON THE EXAM



$$L_i = L_f$$

$$m_b v_{bi} r_{bi} + I_p \omega_i = m_b v_{bf} r_{bf} + I_p \omega_f$$

$$\Rightarrow m_b r_b (v_i - v_f) = I_p \omega_f$$

$$(1 \text{ kg})(3 \text{ m})(10 \text{ m/s} - 4 \text{ m/s}) = (21 \text{ kgm}^2) \omega_f$$

$$\Rightarrow \omega_f = +2 \text{ rad/sec}$$

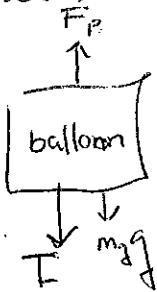
11) a)  $PV = nRT$

$$\Rightarrow n = \frac{PV}{RT}$$

$$= \frac{(10^5 \text{ N/m}^2)(0.1 \text{ m}^3)}{(8.31)(300)}$$

$$= 4.0$$

b) ~~Since the~~ We need to find the tension in the string first.



$$\Rightarrow \sum F = F_p - T - mg = mg(0)$$

$$\Rightarrow T = F_p - mg$$

↳ since it is moving at constant velocity

$$= (1000 \text{ kg/m}^3)(0.1 \text{ m}^3)g - (2 \text{ kg})g$$

$$= 980 \text{ N}$$

(11) b) (cont.)  $T = 980\text{N}$

$$Y = \frac{T/A}{\Delta L/L} = \frac{(980\text{N}/0.05\text{m}^2)}{0.0002} = \boxed{9.8 \times 10^7 \text{ N/m}^2}$$

c)  $P_{\text{Tot}} = P_{\text{atm}} + P_{\text{gauge}} = P_{\text{atm}} + \rho gh$   
 $= 10^5 \text{ N/m}^2 + (1000 \text{ kg/m}^3)(10 \text{ m/s}^2)(90 \text{ m})$   
 $= 10^6 \text{ N/m}^2$

$$P_1 V_1 = P_2 V_2$$
$$\Rightarrow V_2 = \frac{P_1}{P_2} V_1$$
$$= \left( \frac{10^5}{10^6} \right) 0.1 \text{ m}^3$$
$$= 0.01 \text{ m}^3$$

Since the velocity is still constant

$$\Sigma F_y = F_B - T - mg = m a$$

$$\Rightarrow T = F_B - mg$$

$$= (1000 \text{ kg/m}^3)(0.01 \text{ m}^3)g - 2 \text{ kg}g$$
$$= \boxed{80\text{N}}$$

$$(12) Q = L_f m = L_f \rho_{ice} (lA)$$

↳ Latent heat of fusion is heat needed per unit mass

$$\frac{Q}{L} = (K_{ice}) \left( \frac{A}{L} \right) \Delta T$$

↳ temp difference between the two sides of the ice

↳ how much heat makes it through the ice per time

↳ thickness of ice

$$\Rightarrow \frac{L_f \rho_{ice} (lA)}{t} = K_{ice} \left( \frac{A}{L} \right) \Delta T$$

$$\Rightarrow l = (K_{ice}) (\Delta T) \left( \frac{\rho_{ice} t}{\rho_{ice} L \cdot L_f} \right)$$

$$= 0.0026 \text{ m}$$

$$(13) a) \omega = \sqrt{k/m}$$

$$= \sqrt{\frac{200 \text{ N/m}}{2.0 \text{ kg}}} = \sqrt{100} \text{ rad/sec} = 10 \text{ rad/sec}$$

↳ mass of scale + block

$$b) x(t) = y_0 \cos(\omega t - \phi)$$

since it starts at rest,  $\phi = 0$

$$v(t) = -y_0 \omega \sin(\omega t)$$

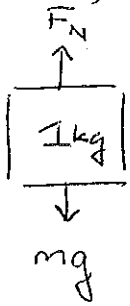
$$v(t) = -\omega y_0 \sin(\omega t)$$

$$a(t) = -\omega^2 y_0 \cos(\omega t)$$

$$= -(10 \text{ Hz})^2 (0.2 \text{ m})$$

$$= 20 \text{ m/s}^2$$

(13) (cont.) c)  $T = \frac{4\pi}{\omega} \Rightarrow \frac{T}{4} = \frac{\pi}{2\omega}; \quad a = -\omega^2 y_0 \cos(\omega(\pi/2\omega)) = 0$



$$\sum F_y = F_z - mg = m(a)$$

$$\Rightarrow F_z = mg$$

$$= (1 \text{ kg})(10 \text{ m/s}^2)$$

$$= 10 \text{ N}$$

d) ~~We want to know if  $F_N = 0$  the acceleration is such that  $F_N = 0$ , is is the scale's case~~

d) We want to know if the scale is ever accelerating away from the block faster than gravity is pulling it down. The math, does

$-10 \text{ m/s}^2 = a(t)$  have a solution

$$-10 \text{ m/s}^2 = -\omega^2 y_0 \cos(\omega t)$$

$$= (100 \text{ s}^{-2})(0.2 \text{ m}) \cos(\omega t)$$

$$\Rightarrow \cos(\omega t) = -0.5$$

So this has a solution at

$$\omega t = 2.09$$

$$\Rightarrow t = 2.09 / 10 \text{ Hz} = \boxed{0.209 \text{ sec}}$$



④ a)  ~~$PV = nRT$~~   $PV = nRT$   
 $\Rightarrow \frac{PV}{T} = nR$

Thus  $T$  is at max when  $PV$  is at max, and min when  $PV$  is min.

$$T_A = \frac{P_A V_A}{nR}$$

$$= \frac{(16600 \text{ N/m}^2)(0.5 \text{ m}^3)}{(2 \text{ mol})(8.314)}$$

$$= 500 \text{ K}$$

$$T_D = \frac{(8300 \text{ N/m}^2)(0.5 \text{ m}^3)}{(2 \text{ mol})(8.314)}$$

$$= 250 \text{ K}$$

$$\Rightarrow \Delta T = 250 \text{ K}$$

b)  $B \rightarrow C$ :  ~~$Q = n C_V R \Delta T$~~   
 (constant volume)  ~~$= (2)(5/2)(8.3)(250)$~~   
 ~~$= -10,375 \text{ J}$~~

~~$\frac{P_A V_A}{T_A} = \frac{P_B V_B}{T_B} \Rightarrow \left(\frac{T_B}{T_A}\right) \left(\frac{V_A}{V_B}\right) P_A = P_B$~~

~~$P_B = \left(\frac{250 \text{ K}}{500 \text{ K}}\right) \left(\frac{0.5 \text{ m}^3}{2.0 \text{ m}^3}\right) (16,600 \text{ N/m}^2) = 2075 \text{ N/m}^2$~~

\* Don't forget to actually make table \*

$$\textcircled{14} \text{ (cont.) b) } B \rightarrow C: Q = n C_p R \Delta T$$

(cons. pressure)

$$= (2) \left(\frac{5}{2}\right) (8.3) (250)$$
$$= 10,350 \text{ J}$$

$$P_A V_A = P_B V_B \Rightarrow P_B = P_A \left(\frac{V_A}{V_B}\right)$$
$$= (16600 \text{ N/m}^2) \left(\frac{0.5 \text{ m}^3}{2.0 \text{ m}^3}\right)$$
$$= \underline{4150 \text{ N/m}^2}$$

$$W_{\text{done on sys}} = -P dV = -P \Delta V$$

$\hookrightarrow$  this equality only holds if  $P$  is cons.

$$= (-4150 \text{ N/m}^2) (1 \text{ m}^3 - 2 \text{ m}^3) = 4150 \text{ J}$$

$$\Delta E_{\text{TH}} = Q + W_{\text{DOS}} = -6200 \text{ J}$$

$$C \rightarrow D: \Delta E_{\text{TH}} = 0; W_{\text{DOS}} = -P dV = -\frac{nRT}{V} dV = -nRT \ln\left(\frac{V_f}{V_i}\right)$$

(isothermal)

$$= (-2)(8.3)(250) \ln\left(\frac{0.5}{1}\right)$$
$$= 2880 \text{ J}$$

$$\Rightarrow Q = -2880 \text{ J}$$

$$D \rightarrow A: Q = n C_v R \Delta T$$

(cons volume)

$$= (2) \left(\frac{3}{2}\right) (8.3) (250)$$
$$= 6200 \text{ J}$$

$$W_{\text{DOS}} = P dV = 0 \text{ [since } V_{\text{cons}}]$$

$$\Rightarrow \Delta E_{\text{TH}} = 6200 \text{ J}$$